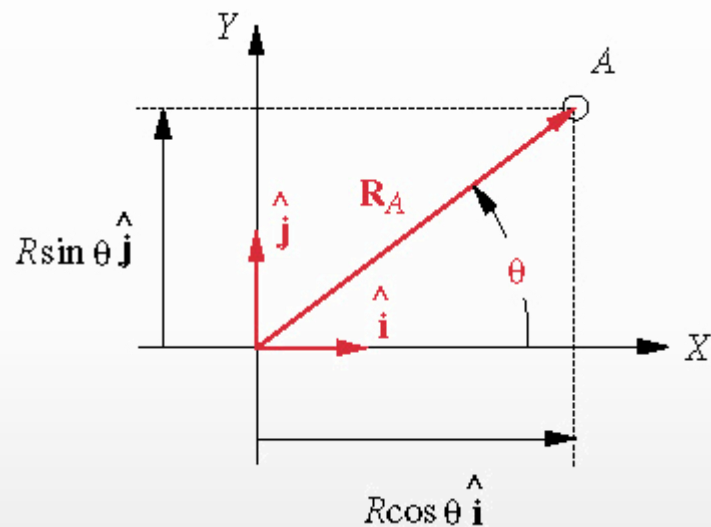


Kinematics & Dynamics of Linkages

Lecture 11 – Analytical Linkage Synthesis

Algebraic analysis

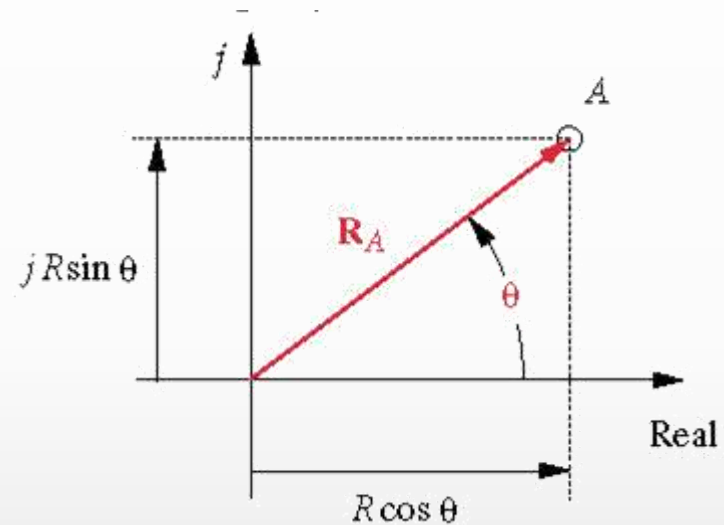


Polar form:

$$| \mathbf{R}_A | @ \theta$$

Cartesian form:

$$R \cos \theta \hat{\mathbf{i}}, R \sin \theta \hat{\mathbf{j}}$$



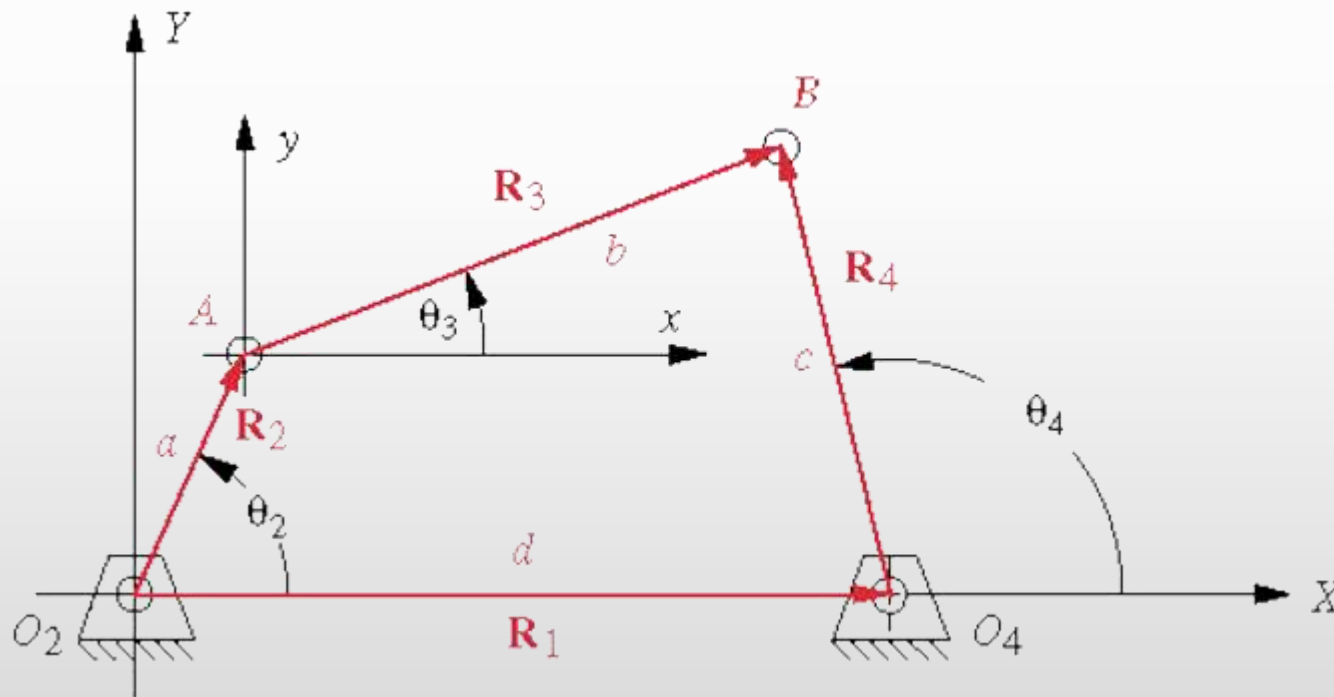
Polar form: $R e^{j\theta}$

Cartesian form: $R \cos \theta + j R \sin \theta$

$$R = | \mathbf{R}_A |$$

Vector Loop Approach

$$\mathbf{R}_2 + \mathbf{R}_3 - \mathbf{R}_4 - \mathbf{R}_1 = \mathbf{0}$$



Equations

$$R_2 + R_3 - R_4 - R_1 = 0$$

Real part

$$a \cos\theta_2 + b \cos\theta_3 - c \cos\theta_4 - d = 0$$

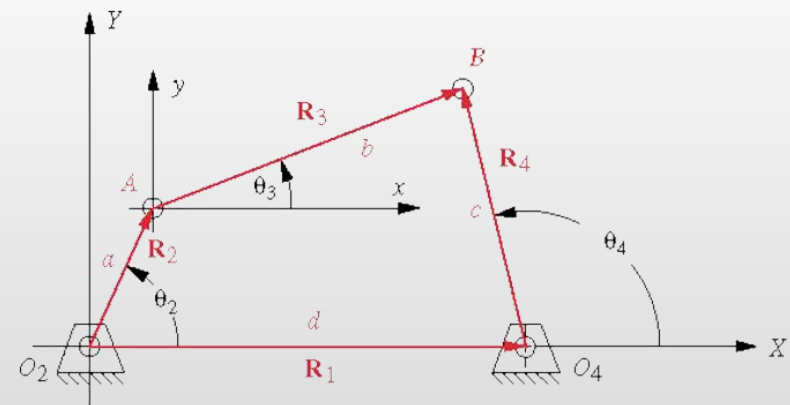
Imaginary part

$$a \sin\theta_2 + b \sin\theta_3 - c \sin\theta_4 = 0$$

Goal is to find

$$\theta_3 = f(a, b, c, d, \theta_2)$$

$$\theta_4 = g(a, b, c, d, \theta_2)$$



4Bar Solution Derivation

Isolate θ_3

$$b \cos\theta_3 = -a \cos\theta_2 + c \cos\theta_4 + d \quad b \sin\theta_3 = -a \sin\theta_2 + c \sin\theta_4$$

Square both sides

$$(b \cos\theta_3)^2 = (-a \cos\theta_2 + c \cos\theta_4 + d)^2 \quad (b \sin\theta_3)^2 = (-a \sin\theta_2 + c \sin\theta_4)^2$$

Add the 2 expressions

$$b^2 (\cos^2\theta_3 + \sin^2\theta_3) = (-a \cos\theta_2 + c \cos\theta_4 + d)^2 + (-a \sin\theta_2 + c \sin\theta_4)^2$$

So this yields (using $\cos^2\theta_3 + \sin^2\theta_3 = 1$)

$$b^2 = (-a \cos\theta_2 + c \cos\theta_4 + d)^2 + (-a \sin\theta_2 + c \sin\theta_4)^2$$

4Bar Solution Derivation

Multiply & combine like terms

$$b^2 = a^2 + c^2 + d^2 - 2ad \cos\theta_2 + cd \cos\theta_4 - 2ac \cos(\theta_2 - \theta_4)$$

Divide both sides by 2ac

$$- (-b^2 + a^2 + c^2 + d^2) / 2ac + (d/c) \cos\theta_2 - (d/a) \cos\theta_4 = -\cos(\theta_2 - \theta_4)$$

To simplify, define

$$k_1 = (d/a)$$

$$k_2 = (d/c)$$

$$k_3 = (a^2 - b^2 + c^2 + d^2) / 2ac$$

Yields Freudenstein's equation

$$k_1 \cos\theta_4 - k_2 \cos\theta_2 + k_3 = \cos(\theta_2 - \theta_4) = \cos\theta_2 \cos\theta_4 + \sin\theta_2 \sin\theta_4$$

4Bar Solution Derivation

Yields Freudenstein's equation

$$k_1 \cos \theta_4 - k_2 \cos \theta_2 + k_3 = \cos \theta_2 \cos \theta_4 + \sin \theta_2 \sin \theta_4$$

From Trigonometric relations:

$$\sin \theta_4 = (2 \tan(\theta_4/2)) / (1 + \tan^2(\theta_4/2))$$

$$\cos \theta_4 = (1 - \tan^2(\theta_4/2)) / (1 + \tan^2(\theta_4/2))$$

Substituting $A \tan^2(\theta_4/2) + B \tan(\theta_4/2) + C = 0$

where

$$A = \cos \theta_2 - k_1 - k_2 \cos \theta_2 + k_3$$

$$B = -2 \sin \theta_2$$

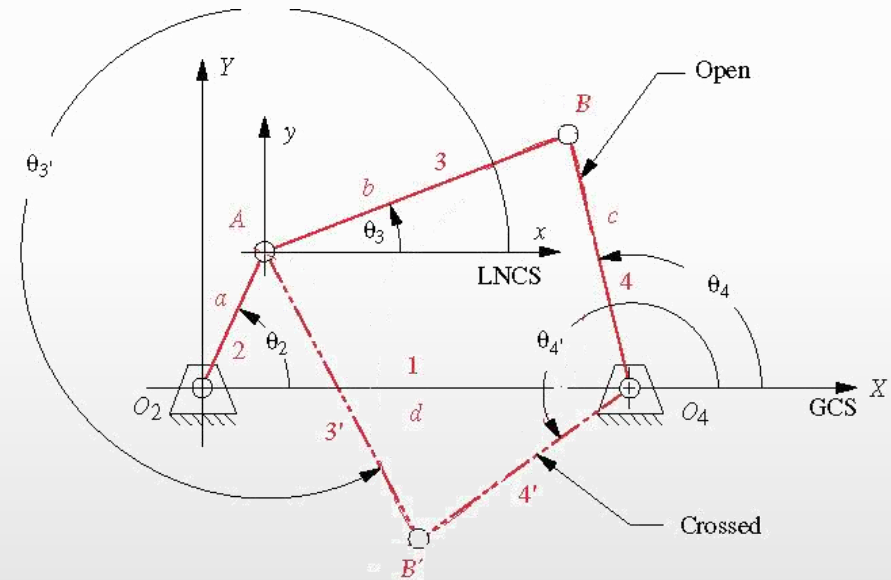
$$C = k_1 - (k_2 + 1) \cos \theta_2 + k_3$$

4Bar Solution Derivation

Solution of $A \tan^2(\theta_4/2) + B \tan(\theta_4/2) + C = 0$

$$\tan\left(\frac{\theta_4}{2}\right) = \frac{-B \pm \sqrt{(B^2 - 4AC)}}{2A}$$

$$\theta_4 = 2 \tan^{-1}\left(\frac{-B \pm \sqrt{(B^2 - 4AC)}}{2A}\right)$$



We have **2 solutions** because we have 2 possible linkage configurations (**Open** and **Crossed**)

4Bar Solution Derivation

Solving for θ_3 :

Original loop equations

$$a \cos\theta_2 + b \cos\theta_3 - c \cos\theta_4 - d = 0 \quad a \sin\theta_2 + b \sin\theta_3 - c \sin\theta_4 = 0$$

Isolate θ_4

$$c \cos\theta_4 = a \cos\theta_2 + b \cos\theta_3 - d \quad c \sin\theta_4 = a \sin\theta_2 + b \sin\theta_3$$

Square both sides and add the equations to eliminate θ_4

$$k_1 \cos\theta_3 - k_4 \cos\theta_2 + k_5 = \cos(\theta_2 - \theta_3) = \cos\theta_2 \cos\theta_3 + \sin\theta_2 \sin\theta_3$$

Where

$$k_1 = (d/a)$$

$$k_4 = (d/b)$$

$$k_5 = (c^2 - d^2 - a^2 - b^2)/2ab$$

4Bar Solution Derivation

Similarly, we may derive a quadratic equation for $\tan(\theta_3/2)$

$$D \tan^2(\theta_3/2) + E \tan(\theta_3/2) + F = 0$$

Where

$$D = \cos\theta_2 - k_1 + k_4 \cos\theta_2 + k_5$$

$$E = -2\sin\theta_2$$

$$F = k_1 + (k_4 - 1)\cos\theta_2 + k_5$$

$$k_1 = (d/a)$$

$$k_4 = (d/b)$$

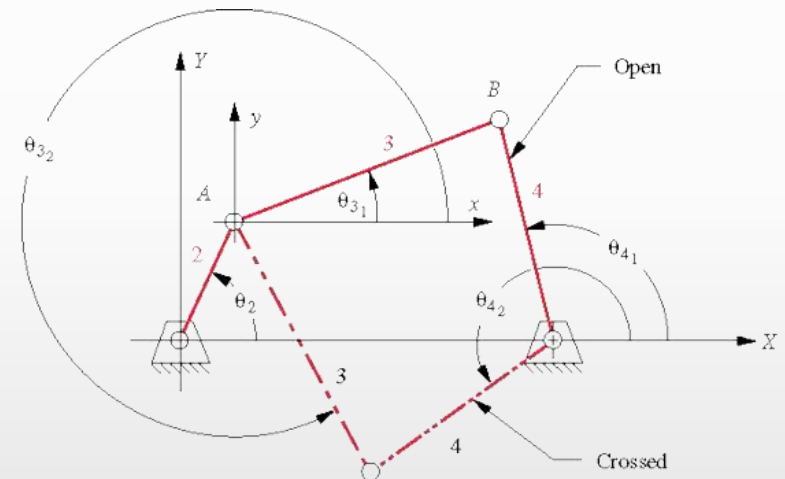
$$k_5 = (c^2 - d^2 - a^2 - b^2)/2ab$$

4Bar Solution Derivation

The solution to the quadratic is

$$\tan\left(\frac{\theta_3}{2}\right) = \frac{-E \pm \sqrt{(E^2 - 4DF)}}{2D}$$

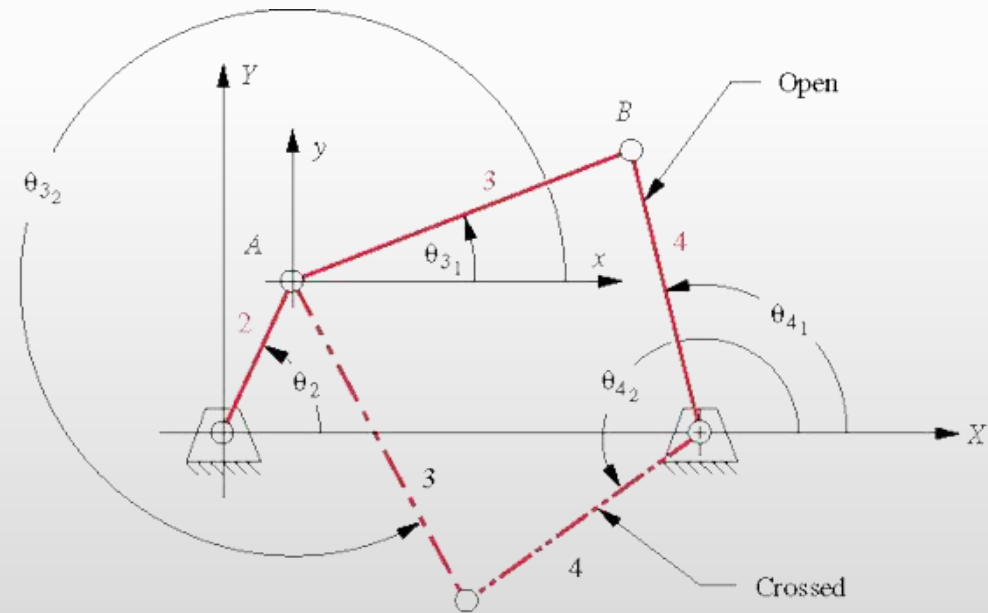
$$\theta_3 = 2 \tan^{-1}\left(\frac{-E \pm \sqrt{(E^2 - 4DF)}}{2D}\right)$$



Again, **2 solutions** for **open & crossed** linkages

Example

- Link 1 = 8"
- Link 2 = 5"
- Link 3 = 8"
- Link 4 = 6"
- $\theta_2 = 75^\circ$
- Calculate θ_3 & θ_4 ??



Example - Solution

To determine θ_3 & θ_4 first calculate the constants

$$k_1 = (d/a) = 1.6$$

$$k_2 = (d/c) = 1.333$$

$$k_3 = (a^2 - b^2 + c^2 + d^2)/2ac = 1.017$$

$$k_4 = (d/b) = 1.0$$

$$k_5 = (c^2 - d^2 - a^2 - b^2)/2ab = -1.463$$

$$A = \cos\theta_2 - k_1 - k_2\cos\theta_2 + k_3 = -0.669$$

$$B = -2\sin\theta_2 = -1.932$$

$$C = k_1 - (k_2 + 1)\cos\theta_2 + k_3 = 2.013$$

$$D = \cos\theta_2 - k_1 + k_4\cos\theta_2 + k_5 = -2.545$$

$$E = -2\sin\theta_2 = -1.932$$

$$F = k_1 + (k_4 - 1)\cos\theta_2 + k_5 = 0.137$$

a = length of link 2

b = length of link 3

c = length of link 4

d = length of link 1

Example - Solution

The solution of θ_4 :

$$\theta_4 = 2 \tan^{-1} \left(\frac{-B \pm \sqrt{(B^2 - 4AC)}}{2A} \right) = 78.2^\circ \text{ or } -149.7^\circ$$

The solution of θ_4 :

$$\theta_3 = 2 \tan^{-1} \left(\frac{-E \pm \sqrt{(E^2 - 4DF)}}{2D} \right) = 7.5^\circ \text{ or } -79.0^\circ$$

Slider-Cranks

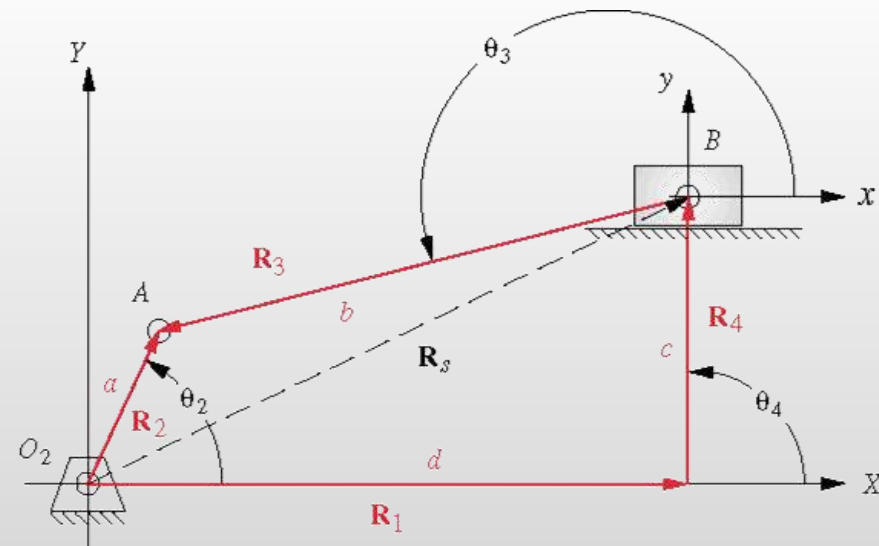
Vector Loop Approach

Given:

- Lengths of links 2 & 3
- Offset height
- Input angle θ_2

Find:

- Length of link 1
- Angle θ_3



Slider-Cranks

Vector loop equation

$$\mathbf{R}_2 - \mathbf{R}_3 - \mathbf{R}_4 - \mathbf{R}_1 = \mathbf{0}$$

Projections: (x and y axis)

$$a \cos\theta_2 - b \cos\theta_3 - c \cos\theta_4 - d = 0$$

$$a \sin\theta_2 - b \sin\theta_3 - c \sin\theta_4 = 0$$

Where

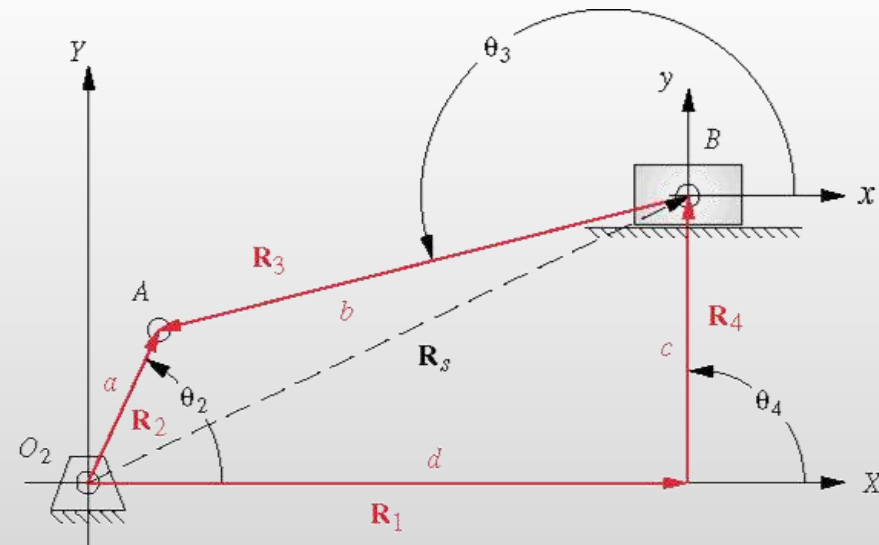
a, b and c are known

θ_2 is given

$\theta_4 = 90^\circ$

Solution

$$d = a \cos\theta_2 - b \cos\theta_3$$



Slider-Cranks

Solve for θ_3 using the equation (projection on y axis)

$$a \sin\theta_2 - b \sin\theta_3 - c \sin\theta_4 = 0$$

Solution ($\pm 90^\circ$ of \sin^{-1})

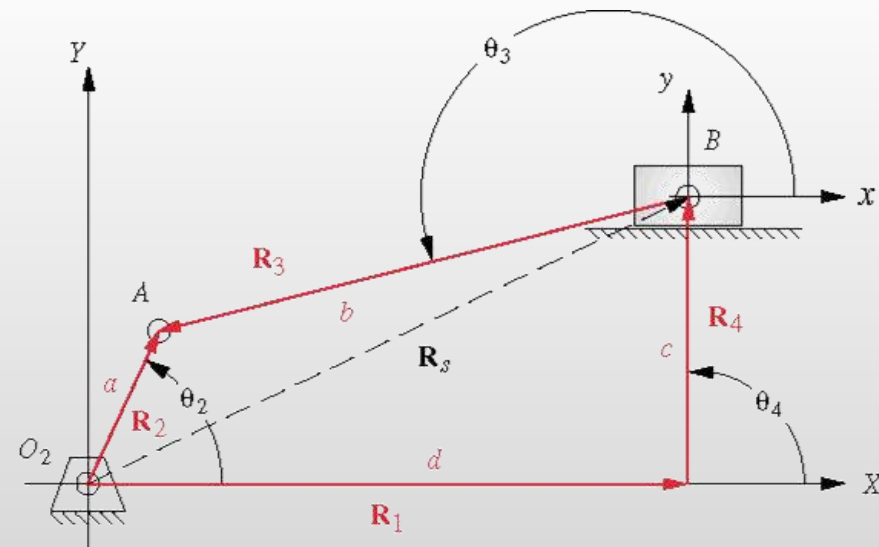
1st configuration

$$\theta_{3_1} = \sin^{-1}\left(\frac{a \sin\theta_2 - c}{b}\right)$$

2nd configuration

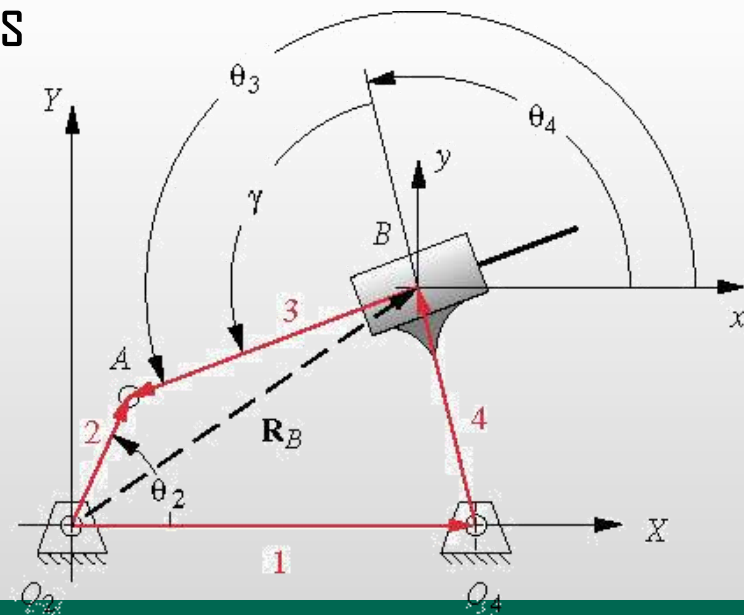
$$\theta_{3_2} = \sin^{-1}\left(-\frac{a \sin\theta_2 - c}{b}\right) + \pi$$

$$d = a \cos\theta_2 - b \cos\theta_3$$



Inverted Slider-Cranks

- Sliding joint between links 3 & 4 at point B
- γ does not have to be 90°
- All slider-crank mechanisms will have at least one link whose effective length between joints will vary as the linkage moves



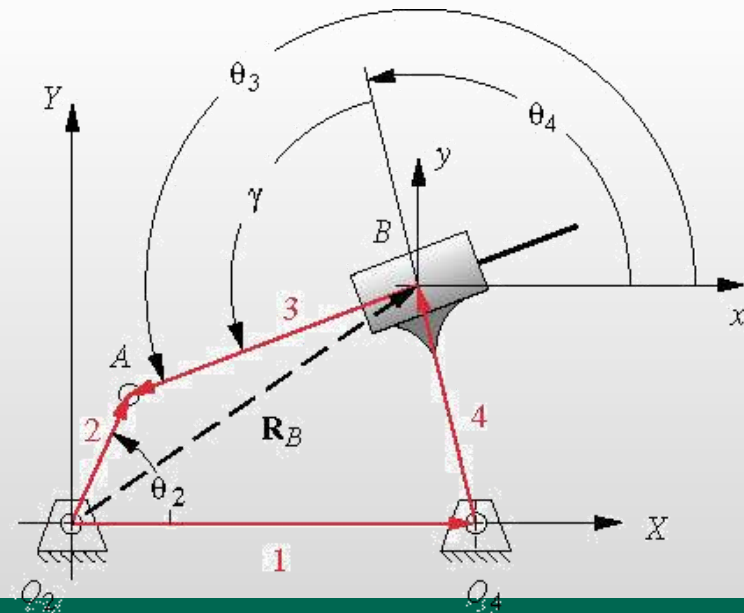
Inverted Slider-Cranks

- **Vector Loop Approach**
- Length of link 3 (b) changes with time
 - b is unknown
 - θ_4 is unknown
 - θ_3 is unknown

- Relationship between θ_3 and θ_4

$$\theta_3 = \theta_4 + \gamma$$

- + for an open linkage
- for a crossed linkage



Inverted Slider-Cranks

Vector loop equation: $R_2 - R_3 - R_4 - R_1 = 0$

Original loop equations

$$a \cos\theta_2 - b \cos\theta_3 - c \cos\theta_4 - d = 0$$

$$a \sin\theta_2 - b \sin\theta_3 - c \sin\theta_4 = 0$$

Where a , d and c are known, θ_2 is given, θ_4 , θ_3 and b to be calculated

Use the second equation to solve for b

$$b = (a \sin\theta_2 - c \sin\theta_4) / \sin\theta_3$$

Substituting into the first equation

$$a \cos\theta_2 - ((a \sin\theta_2 - c \sin\theta_4) / \sin\theta_3) \cos\theta_3 - c \cos\theta_4 - d = 0$$

Inverted Slider-Cranks

Rearrange equation

$$a \cos\theta_2 - ((a \sin\theta_2 - c \sin\theta_4) / \sin\theta_3) \cos\theta_3 - c \cos\theta_4 - d = 0$$

Using: $\theta_3 = \theta_4 \pm \gamma$

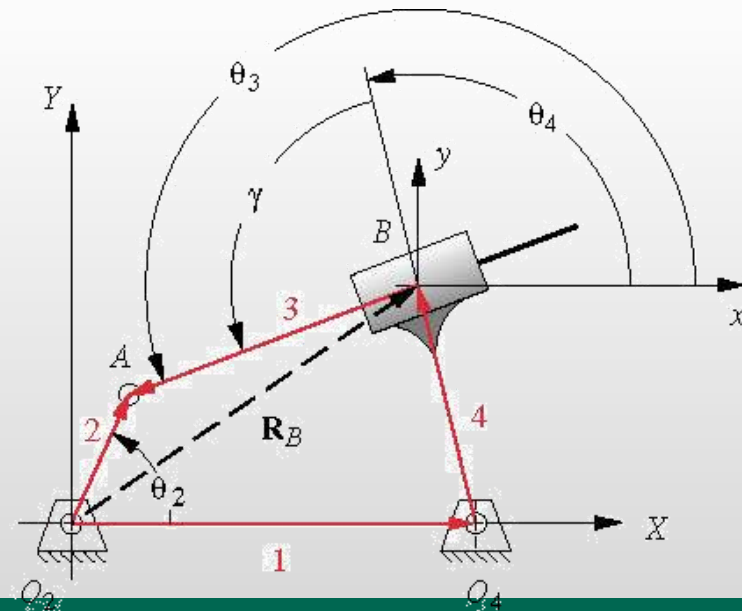
Yields: $P \sin\theta_4 + Q \cos\theta_4 + R = 0$

Where

$$P = a \sin\theta_2 \sin\gamma + (a \cos\theta_2 - d) \cos\gamma$$

$$Q = -a \sin\theta_2 \cos\gamma + (a \cos\theta_2 - d) \sin\gamma$$

$$R = -c \sin\gamma$$



Inverted Slider-Cranks

Using half angles:

$$P \sin\theta_4 + Q \cos\theta_4 + R = 0$$

$$(R - Q) \tan^2(\theta_4/2) + 2P \tan(\theta_4/2) + (Q + R) = 0$$

Let $S = R - Q$

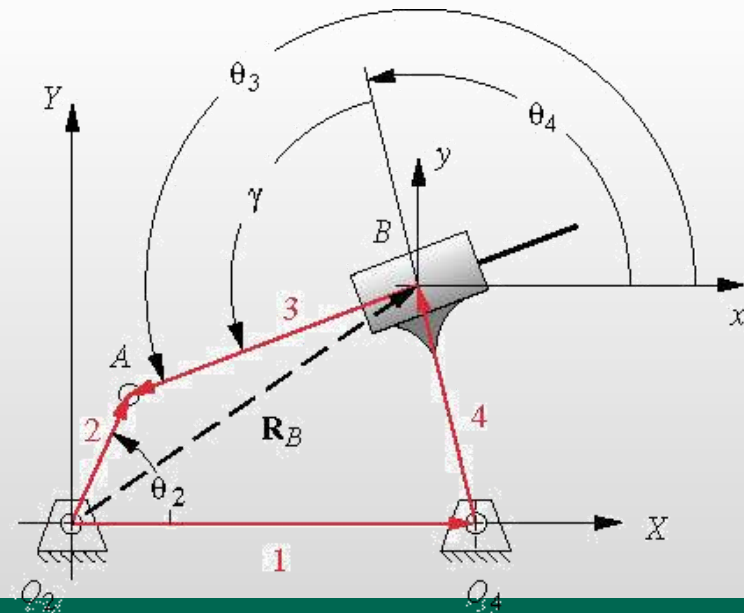
$$T = 2P$$

$$U = Q + R$$

Solution

$$\theta_4 = 2 \arctan \left(\frac{-T \pm (T^2 - 4SU)^{.5}}{2S} \right)$$

This has both **open** & **crossed** solutions



Inverted Slider-Cranks

Having θ_4 solve for θ_3 and b

Open configuration

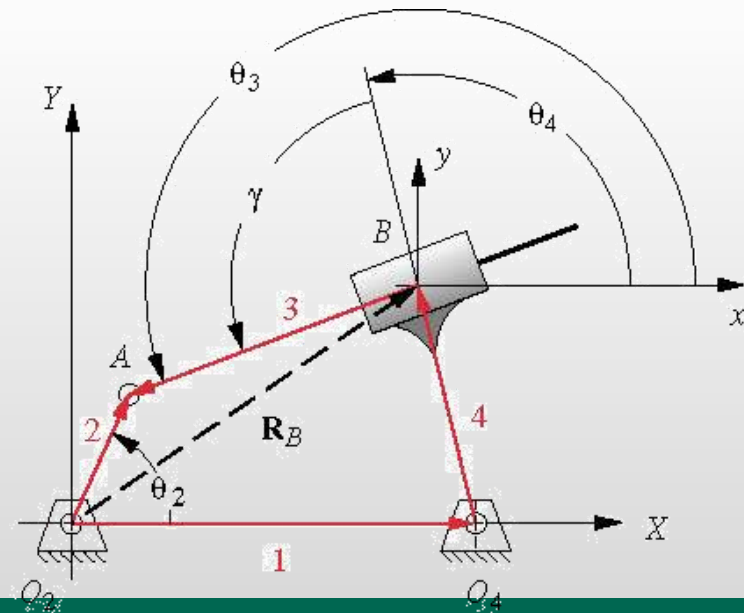
$$\theta_3 = \theta_4 + \gamma$$

$$b = (a \sin\theta_2 - c \sin\theta_4) / \sin\theta_3$$

Crossed configuration

$$\theta_3 = \theta_4 - \gamma$$

$$b = (a \sin\theta_2 - c \sin\theta_4) / \sin\theta_3$$

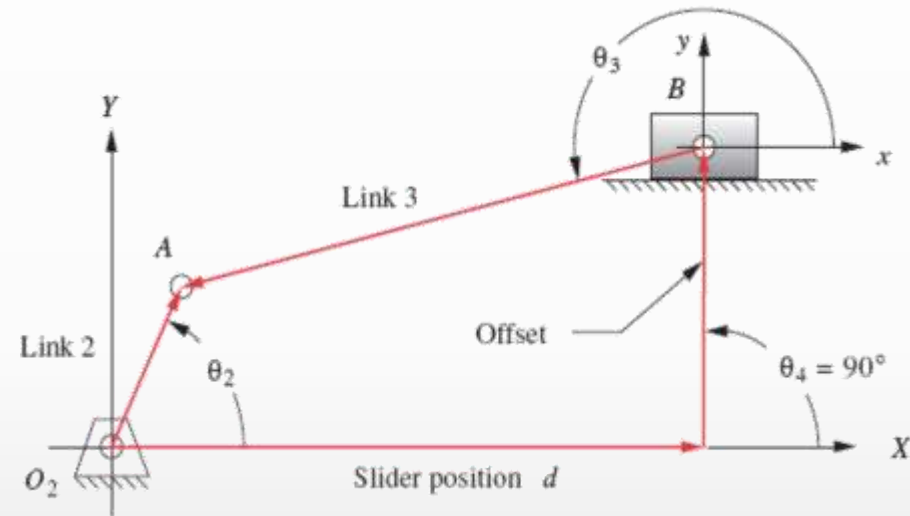


Example – Slider Crank

Given:

- Link 2 = 1.4"
- Link 3 = 4"
- Offset = 1"
- $\theta_2 = 45^\circ$

Find: θ_3 & d



Example – Slider Crank Solution

Use formulas derived before

1st configuration (crossed)

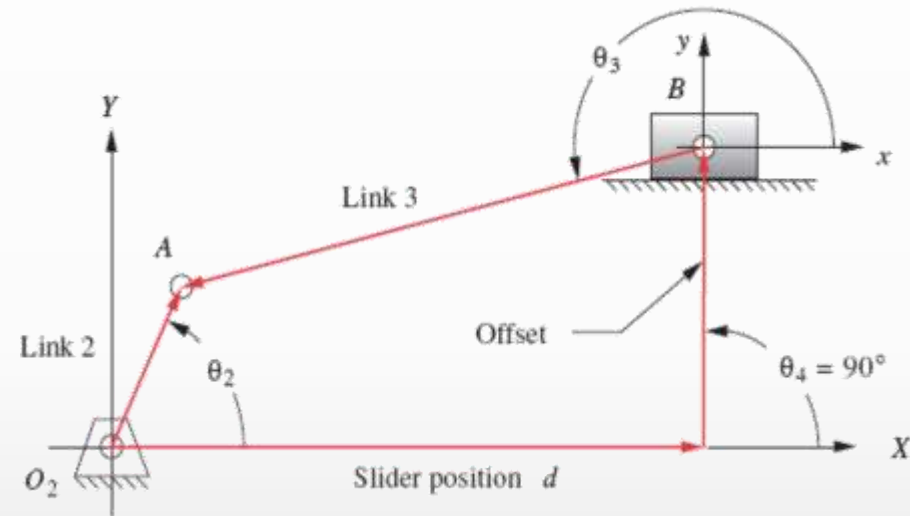
$$\theta_{3_1} = \sin^{-1}\left(\frac{a \sin \theta_2 - c}{b}\right) = -0.14^\circ$$

$$d = a \cos \theta_2 - b \cos \theta_3 = -3.01 \text{ in}$$

2nd configuration (open)

$$\theta_{3_2} = \sin^{-1}\left(-\frac{a \sin \theta_2 - c}{b}\right) + \pi = 180.14^\circ$$

$$d = a \cos \theta_2 - b \cos \theta_3 = 4.99 \text{ in}$$



a = length of link 2

b = length of link 3

c = length of link 4

d = length of link 1

Solution

